



BANGLADESH UNIVERSITY OF ENGINEERING & TECHNOLOGY
DEPARTMENT OF MECHANICAL ENGINEERING

Course No.: ME 346 (Mechanics of Machinery Sessional)

Problem Class: 01 (C1)

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Class Problems:

1. A spring-mass system has a natural frequency of 10 Hz. When the spring constant is reduced by 800 N/m, the frequency is altered by 45 percent. Find the mass and spring constant of the original system.
2. A helicopter landing gear consists of a metal framework rather than the coil spring based suspension system used in a fixed-wing aircraft. The vibration of the frame in the vertical direction can be modeled by a spring made of a slender bar, where the helicopter is modeled as ground. Here $l = 0.4$ m, $E = 20 \times 10^{10}$ N/m², and $m = 100$ kg. Calculate the cross-sectional area that should be used if the natural frequency is to be $f_n = 500$ Hz.
3. A shaft of length 0.75 m, supported freely at the ends, is carrying a body of mass 90 kg at 0.25m from one end. Find the natural frequency of transverse vibration. Assume $E = 200$ GN/m² and shaft diameter = 50mm.
4. A four-wheeled trolley car of total mass 2000 kg running on rails of 1.6 m gauge, rounds a curve of 30 m radius at 54 km/h. The track is banked at 8°. The wheels have an external diameter of 0.7 m and each pair with axle has a mass of 200 kg. The radius of gyration for each pair is 0.3 m. The height of centre of gravity of the car above the wheel base is 1 m. Determine, allowing for centrifugal force and gyroscopic couple actions, the pressure on each rail.

Solution

Problem # 1

A spring-mass system has a natural frequency of 10 Hz. When the spring constant is reduced by 800 N/m, the frequency is altered by 45 percent. Find the mass and spring constant of the original system.

Given,

$$\omega_n = 10 \text{ Hz} = 62.832 \frac{\text{rad}}{\text{s}}$$

$$(\omega_n)_{\text{new}} = 0.55 \omega_n = 34.5576 \frac{\text{rad}}{\text{s}}$$

Now,

$$\omega_n = \sqrt{\frac{k}{m}}$$

Thus,

$$\frac{(\omega_n)_{\text{new}}}{\omega_n} = 0.55$$

$$\Rightarrow \sqrt{\frac{k-800}{k}} = 0.55$$

$$\Rightarrow k = 1146.95 \frac{\text{N}}{\text{m}}$$

$$\therefore \sqrt{m} = \frac{\sqrt{1146.95}}{62.832}$$

$$\Rightarrow m = 0.2905 \text{ kg}$$

Problem # 2

A helicopter landing gear consists of a metal framework rather than the coil spring based suspension system used in a fixed-wing aircraft. The vibration of the frame in the vertical direction can be modeled by a spring made of a slender bar, where the helicopter is modeled as ground. Here $l = 0.4 \text{ m}$, $E = 20 \times 10^{10} \text{ N/m}^2$, and $m = 100 \text{ kg}$. Calculate the cross-sectional area that should be used if the natural frequency is to be $f_n = 500 \text{ Hz}$.

Given,

$$\omega_n = 500 \text{ Hz} = 3142 \frac{\text{rad}}{\text{s}}$$

Now,

$$\omega_n = \sqrt{\frac{k}{m}}$$

Considering a slender bar, the stiffness can be calculated as,

$$F = k\delta$$

$$\Rightarrow k = \frac{F}{\delta} = \frac{F/A \times A}{\epsilon l}$$

$$\Rightarrow k = \frac{\sigma A}{\epsilon l} = \frac{EA}{l}$$

Thus,

$$\omega_n = \sqrt{\frac{k}{m}} = \sqrt{\frac{EA}{lm}}$$

$$\Rightarrow 3142 = \sqrt{\frac{20 \times 10^{10} \times A}{0.4 \times 100}}$$

$$\Rightarrow A = 0.0019 \text{ m}^2 = 19 \text{ cm}^2$$

Problem # 3

A shaft of length 0.75 m, supported freely at the ends, is carrying a body of mass 90 kg at 0.25m from one end. Find the natural frequency of transverse vibration. Assume $E = 200 \text{ GN/m}^2$ and shaft diameter = 50mm.

Given,

$$l = 0.75 \text{ m}$$

$$m = 90 \text{ kg}$$

$$a = 0.25 \text{ m}$$

$$E = 200 \text{ GN/m}^2$$

$$d = 50 \text{ mm} = 0.05 \text{ m}$$

Now,

$$I = \frac{\pi}{64} d^4 = \frac{\pi}{64} \times (0.05)^4 \text{ m}^4$$

$$\Rightarrow I = 0.307 \times 10^{-6} \text{ m}^4$$

And,

$$\delta = \frac{Wa^2b^2}{3EI} = \frac{90 \times 9.81 \times (0.25)^2 \times (0.5)^2}{3 \times 200 \times 10^9 \times 0.307 \times 10^{-6} \times 0.75}$$

$$\Rightarrow \delta = 0.1 \times 10^{-3} \text{ m}$$

Thus,

$$f_n = \frac{1}{2\pi} \sqrt{\frac{g}{\delta}} = \frac{0.4985}{\sqrt{0.1 \times 10^{-3}}} = 49.85 \text{ Hz}$$

Problem # 4

A four-wheeled trolley car of total mass 2000 kg running on rails of 1.6 m gauge, rounds a curve of 30 m radius at 54 km/h. The track is banked at 8° . The wheels have an external diameter of 0.7 m and each pair with axle has a mass of 200 kg. The radius of gyration for each pair is 0.3 m. The height of centre of gravity of the car above the wheel base is 1 m. Determine, allowing for centrifugal force and gyroscopic couple actions, the pressure on each rail.

Given,

$$m_v = 2000 \text{ kg}$$

$$x = 1.6 \text{ m}$$

$$v = 54 \text{ km/h} = 15 \text{ m/s}$$

$$\theta = 8^\circ$$

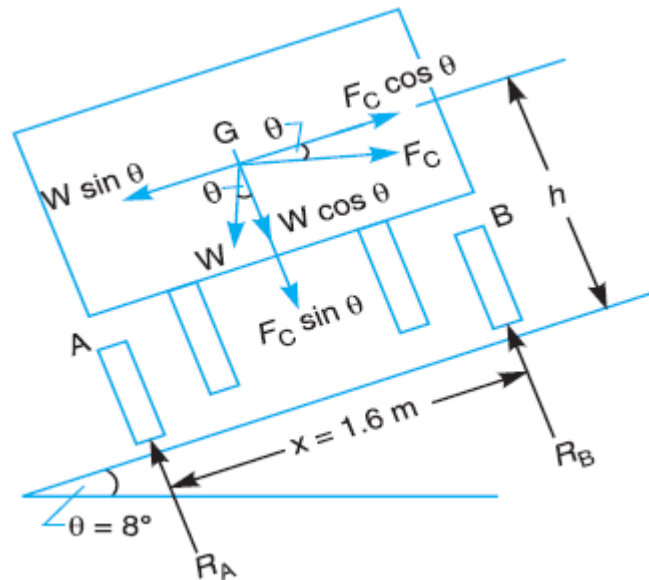
$$d_w = 0.7 \text{ m}$$

$$m_a = 200 \text{ kg}$$

$$h = 1 \text{ m}$$

$$k = 0.3 \text{ m}$$

$$R = 30 \text{ m}$$



Now,

$$\sum F = R_A + R_B = W \cos \theta + F_c \sin \theta = mg \cos \theta + \frac{m_v v^2}{R} \sin \theta$$

$$\Rightarrow R_A + R_B = 2000 \times 9.81 \cos 8^\circ + \frac{2000 \times (15)^2}{30} \sin 8^\circ$$

$$\Rightarrow R_A + R_B = 21,518 \text{ N}$$

Now, taking moments about B,

$$R_A \times x = (W \cos \theta + F_c \sin \theta) \frac{x}{2} + W \sin \theta \times h - F_c \cos \theta \times h$$

$$\Rightarrow R_A = \left[2000 \times 9.81 \cos 8^\circ + \frac{2000 \times (15)^2}{30} \sin 8^\circ \right] \times \frac{1}{2} + 2000 \times 9.81 \sin 8^\circ \times 1 - \frac{2000 \times (15)^2}{30} \cos 8^\circ \times \frac{1}{1.6}$$

$$\Rightarrow R_A = 3,182 \text{ N}$$

$$\therefore R_B = 18,226 \text{ N}$$

Now, angular speed of wheel,

$$\omega_w = \frac{v}{r_w} = \frac{15}{0.35} = 42.86 \frac{\text{rad}}{\text{s}}$$

And, angular speed of precession,

$$\omega_p = \frac{v}{R} = \frac{15}{30} = 0.5 \frac{rad}{s}$$

Thus, gyroscopic couple,

$$C = I \omega_w \cos \theta \times \omega_p = m k^2 \omega_w \cos \theta \omega_p$$
$$\Rightarrow C = 200 \times (0.3)^2 \times 42.86 \cos 8^\circ \times 0.5 = 382 \text{ N} - m$$

Pressure due to the gyroscopic couple,

$$P = \frac{C}{x} = \frac{382}{1.6} = 238.75 \text{ N}$$

Pressure on the inner rail,

$$P_I = R_A - P = 2,943.25 \text{ N}$$

Pressure on the outer rail,

$$P_O = R_B + P = 18,574.75 \text{ N}$$
